

**New Folder Name** Beam Tube Design

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LIGO - BEAM TUBE DESIGN

- INPUT VARIABLES

Tube outside diameter, Do = 49.004 in  
 Beam Tube section length, L = 19.812 m = 65.000 ft  
 Beam Tube Span length, Lsp = 18.9484 m = 62.167 ft  
 Tube thickness, t = 0.127 in  
 Insulation Density, Deni = 24 kg/m = 16.127 lbs/ft

Vacuum Stiffener thickness, ts = 0.1875 in  
 Vacuum Stiffener width, ws = 1.75 in  
 Vacuum Stiffener spacing, Ls = 29.84 in  
 Support Stiffener thickness, tss = 0.375 in  
 Support Stiffener width, wss = 4 in

Mod. of Elast. @ ambient, Ea = 28,300 ksi Table TM-1 @ 70  
 Mod. of Elast. @ 302 degrees, Eb = 27,000 ksi Table TM-1, Page 664  
 Coefficient of expansion, e = 9E-06 in/in/F Average from 70 to 300 degrees F

Anchor bolt spacing, Abs = 30 in  
 C. line height of tube, H = 42.000 in  
 Support Collar / Saddle width, b = 20.000 in

Expansion Joint O.D., De = 53.75 in  
 Expansion Joint I.D., Del = 48.75 in  
 E.J. Concentricity Error, CE = 0.1875 in  
 Expansion joint axial spring rate, Kej = 9147.6 lbs/in = 8316 \* 1.1  
 Spring rate variation, Eej = 10.0%  
 Spring Rate, K = 10062 lbs/in = Kej \* ( 1 + Eej )

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	DATE	DATE	DATE	DATE	
LIGO - BEAM TUBE DESIGN Configuration 3, K = 8316 + 20% Maximum Differential Settlements					6.5

## - TUBE WEIGHT &amp; PROPERTIES

Inside Diameter, Di = 48.75 in  
 Area, A =  $19.501 \text{ in}^2 = \text{PI}() * (\text{Do}^2 - \text{Di}^2) / 4$   
 Section Modulus, S =  $237.7 \text{ in}^3 = \text{PI}() * (\text{Do}^4 - \text{Di}^4) / 32 / \text{Do}$   
 Moment of inertia, I =  $5823 \text{ in}^4 = \text{PI}() * (\text{Do}^4 - \text{Di}^4) / 64$   
 Radius of gyration, rg =  $17.281 \text{ in} = (I / A)^{0.5}$

Theoretical # of vacuum stiffeners = 27 = Round(L/Ls\*12)  
 Number of stiffeners used, Ns = 25  
 True spacing = 31.200 in = L/(Ns)\*12

Number of support stiffeners, Nss = 1.5 Per section

Shell weight per section = 4357 lbs =  $495 * A / 144 * L$

Weight per vacuum stiffener =  $14.987 \text{ lbs} = \text{PI}() * ((\text{Do} + 2 * \text{ws})^2 - \text{Do}^2) / 4 * \text{Ts} * 495 / 12^3$   
 Weight per support stiffener =  $71.550 \text{ lbs} = \text{PI}() * ((\text{Do} + 2 * \text{wss})^2 - \text{Do}^2) / 4 * \text{Tss} * 495 / 12^3$   
 Stiffener weight per section = 482.01 lbs = Weight Vacuum \* Ns + Weight support \* Nss  
 Estimated Baffle wt / section = 27.5 lbs

Total metal weight, DL = 4867 lbs, or  
 wd = 74.87 lbs/ft

Insulation weight per section = 1048 lbs = Deni \* L

DL + Insulation = 5915 lbs, or  
 wdl = 91.00 lbs/ft

SUBJECT

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 Maximum Differential Settlements

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REFERENCE NO.

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DATE	DATE

SHT 2 OF 29

66

- ALLOWABLE STRESS PER ASME SECTION VIII DIV 1 UG 23(b)

- Allowable Stresses @ 300 Degrees F.

$$\text{Yield Stress, } F_y = \frac{19,200 \text{ psi}}{13000 \text{ psi}}$$

$$\text{Tensile Allowable, } S_h = \frac{13000 \text{ psi}}{0.7}$$

$$\text{Joint Efficiency, } E_t = \frac{0.7}{0.7}$$

Compression Allowable

$$A = 0.000648 = 0.125 / (Ro/t)$$

$$B = \frac{5900 \text{ psi per Figure HA 3, interpolate to 300 degrees F.}}{5900 \text{ psi per Figure HA 3, interpolate to 300 degrees F.}}$$

Where  $B = F_a = F_{bx} = F_{by} = F_{bxy}$ , local buckling stress

See below for column buckling

Table Y1

Table 1A

UG 23(b)

- Allowable Stresses @ Ambient (100 degrees F)

$$\text{Yield Stress} = \frac{25,000 \text{ psi}}{16,300 \text{ psi}}$$

$$\text{Tensile Allowable, } S_a = \frac{16,300 \text{ psi}}{16,300 \text{ psi}}$$

Compression Allowable

$$A = 0.000648 = 0.125 / (Ro/t)$$

$$B = \frac{7800 \text{ psi per Figure HA 3, 100 degrees F.}}{7800 \text{ psi per Figure HA 3, 100 degrees F.}}$$

Where  $B = F_a = F_{bx} = F_{by} = F_{bxy}$

Table Y1

Table 1A

UG 23(b)

- Allowable Stress Increase for Wind and Seismic  
Allowable increase for wind or seismic is 1.20

- Allowable Axial Stress, Column Buckling per AISC

$$k = \frac{1}{1}$$

$$L = \frac{746 \text{ in} = L_{sp} * 12}{746 \text{ in} = L_{sp} * 12}$$

$$r = \frac{17.28069 \text{ in} = r_g}{17.28069 \text{ in} = r_g}$$

$$kL / r = 43.16958$$

$$C_c = 166.6081 = (2 * \pi^2 * E_b * 1000 / F_y)^{0.5}$$

$$F_a = 10533 \text{ psi} = \frac{(1 - (kL/r)^2 / 2 / C_c^2) * F_y}{(5/3 + 3 * (kL/r) / 8 / C_c - (kL/r)^3 / 8 / C_c^3)}$$

$$F_a > B, \text{ Thus use } B = 5900 \text{ psi}$$

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LIGO - BEAM TUBE DESIGN  
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Maximum Differential Settlements

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DATE	DATE	DATE	DATE
3/11/94	4-4-94		
SHT			OF
			37

## - CALCULATE WIND LOAD PER ASCE 7-88 (Livingston, LA)

$$F = Qz * Gh * Cf * Af$$

$$Qz = 4.1472 = 0.00256 * Kz * (I * V)^2$$

$$Kz = \frac{0.80}{1.00}, \text{ Assume exposure C}$$

$$V = \frac{45 \text{ mph}}{1.00}$$

$$Gh = \frac{1.32}{0.739}$$

$$Cf = \frac{0.739}{0.739} \quad h / D = 0.0853$$

$$h = (Do / 2 + H) / 12$$

$$D = L$$

$$D' / Do = ws / Do = 0.03571$$

$$Af = 4.084 \text{ Sqft / ft} = Do / 12$$

$$F = 16.53 \text{ lbs / ft}$$

$$F = 1074 \text{ lbs / section}$$

Table 4  
Eq 3  
Table 6  
Table 5  
Figure 1  
Table 8  
Table 12

## - CALCULATE WIND LOAD PER ASCE 7-88 (Hanford, WA)

$$F = Qz * Gh * Cf * Af$$

$$Qz = 4.1472 = 0.00256 * Kz * (I * V)^2$$

$$Kz = \frac{0.80}{1.00}, \text{ Assume exposure C}$$

$$V = \frac{45 \text{ mph}}{1.00}$$

$$Gh = \frac{1.32}{0.739}$$

$$Cf = \frac{0.739}{0.739} \quad h / D = 0.0853$$

$$h = (Do / 2 + H) / 12$$

$$D = L$$

$$D' / Do = ws / Do = 0.03571$$

$$Af = 4.083667 \text{ Sqft / ft} = Do / 12$$

$$F = 16.53 \text{ lbs / ft} = ww$$

$$F = 1074 \text{ lbs / section}$$

Table 4  
Eq 3  
Table 6  
Table 5  
Figure 1  
Table 8  
Table 12

SUBJECT

LIGO - BEAM TUBE DESIGN  
Configuration 3, K = 8316 + 20%  
Maximum Differential Settlements

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SHT

4

OF

19

4.8

## - CALCULATE SEISMIC LOAD PER UBC (Livingston, LA)

$$V = Z * I * C / R_w * W$$

$$Z = 0$$

$$I = 1.0$$

$$C = 2.75$$

$$R_w = 3$$

$$C/R_w = 0.916667 > 0.075 \text{ use: } 0.91667$$

$$W = 91.00 \text{ lbs/ft} = \text{DL} + \text{Insulation}$$

$$V = 0.0500W = 0.05W \text{ minimum per ASCE 7-88, 9.11.2}$$

$$V = 4.55 \text{ Lbs/ft}$$

$$V = 295.8 \text{ Lbs per section}$$

## - CALCULATE SEISMIC LOAD PER ASCE 7-88 (Hanford, WA)

$$V = Z * I * C / R_w * W$$

$$Z = 0.2$$

$$I = 1.0$$

$$C = 2.75$$

$$R_w = 3$$

$$C/R_w = 0.916667 > 0.075 \text{ use: } 0.91667$$

$$W = 91.00 \text{ lbs/ft} = \text{DL} + \text{Insulation}$$

$$V = 0.1833W$$

$$V = 16.68 \text{ Lbs/ft} = \text{wseis}$$

$$V = 1084.4 \text{ Lbs per section}$$

## - CALCULATE SNOW LOAD PER ASCE 7-88 (Hanford, WA only)

$$P_f =$$

$$15.12 = 0.7 * C_e * C_t * I * P_g$$

$$C_e = 0.9 \text{ Little shelter assumed}$$

$$C_t = 1.2 \text{ Unheated}$$

$$I = 1 \text{ Category 1}$$

$$P_g = 20 \text{ psf}$$

Eq 5a

Table 18

Table 19

Table 20

Figure 6

$$P_s =$$

$$9.828 = C_s * P_f$$

$$C_s = 0.65 \text{ Angle is } 35 \text{ degrees, } 90 - (180 - 70) / 2, \text{ Figure 8b}$$

$$\text{Width} = 46.049 \text{ in} = D_o * \sin 70 \text{ Per 7.4.3, slope} > 70 \text{ no load}$$

$$\text{Snow Load per foot, wsn} = 37.71 \text{ lbs/ft} = P_s * \text{Width}$$

$$\text{Snow load per section} = 2451 \text{ lbs}$$

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LIGO - BEAM TUBE DESIGN  
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Maximum Differential Settlements

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REFERENCE NO.  
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SHT 5 OF 29

DATE  
3/11/94DATE  
4-4-94DATE  
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DATE

6.9

## - DETERMINE REACTIONS, STRESSES AND DEFLECTIONS FOR INDIVIDUAL LOAD CASES

## - Nomenclature

Rfx1 = Reaction, fixed support, x direction (horizontal), case 1  
 Rgy2 = Reaction, guided support, y direction (vertical), case 2  
 Rfz1 = Reaction, fixed support, z direction (axial), case 1  
 Mx1 = Moment about the horizontal axis due to vertical loads, Case 1.  
 My2 = Moment about the vertical axis due to horizontal loads, Case 2.  
 fa9c = Stress, axial, case 9, compression, (if tension t is used instead of c)

## - Reactions Based on RISA2D

Fixed support = 7.029 kips, per RISA2D  
 1/2 of guided support =  $\frac{2.41}{11.849}$  kips, per RISA2D  
 Total, two spans = 11.849 kips, per RISA2D

% Fixed support, Kf = 0.593215 (Percentage of 2 spans)  
 % Guided support, Kg = 0.406785

## - Moments based on RISA2D

Maximum Moment = 43480 lb-ft, at fixed support  
 Based on,  $\frac{91}{11.849}$  lbs/ft, DL + Insulation, used in RISA2D

True DL + Insulation = wdi = 91.00  
 Estimated Moment = 43961 lb-ft = wdi \* (Lsp ft)<sup>2</sup> / 8

Moment correction factor, Km = 0.989 = RISA2D moment / Estimated moment

## - Deflections based on RISA2D

Midspan Deflection =  $\frac{0.079}{11.849}$  in  
 Based on,  $\frac{91}{11.849}$  lbs/ft, DL + Insulation, used in RISA2D

True DL + Insulation = wdi = 91.00  
 Estimated Deflection = 0.081 in = wdi \* Lsp<sup>4</sup> / 185 / Eb / I \* 12<sup>3</sup> / 1000

Moment correction factor, Kd = 0.978 = RISA2D deflection / Estimated deflection

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LIGO - BEAM TUBE DESIGN  
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 Maximum Differential Settlements

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DATE	DATE

## REFERENCE NO.

930212

SHT 4 OF 29

6.10

- CASE 1: Dead Load Reactions, Stresses and Deflections  
FIXED SUPPORT

$$\begin{aligned} R_{fx1} &= 0 \\ R_{fy1} &= 5774 \text{ lbs} = K_f * 2 * L * wd \\ R_{fz1} &= 0 \end{aligned}$$

## GUIDED SUPPORT

$$\begin{aligned} R_{gx1} &= 0 \\ R_{gy1} &= 3959 \text{ lbs} = K_g * 2 * L * wd \end{aligned}$$

## MOMENTS &amp; BENDING STRESS

$$\begin{aligned} M_{x1} &= 35774 \text{ lb-ft} = wd * (Lsp)^2 / 8 * Km \\ f_{bx1} &= 1806 \text{ psi} = M_{x1} * 12 / S \end{aligned}$$

$$\begin{aligned} M_{y1} &= 0 \\ f_{by1} &= 0 \text{ psi} = M_{y1} * 12 / S \end{aligned}$$

## MIDSPAN DEFLECTION

$$Dely1 \text{ (amb)} = 0.062 \text{ in} = wd * Lsp^4 / 185 / Ea / I * 12^3 / 1000 * Kd$$

- CASE 2: Dead Load plus Insulation Reaction, Stresses and Deflections  
FIXED SUPPORT

$$\begin{aligned} R_{fx2} &= 0 \\ R_{fy2} &= 7018 \text{ lbs} = K_f * 2 * L * wdi \\ R_{fz2} &= 0 \end{aligned}$$

## GUIDED SUPPORT

$$\begin{aligned} R_{gx2} &= 0 \\ R_{gy2} &= 4812 \text{ lbs} = K_g * 2 * L * wdi \end{aligned}$$

## MOMENTS &amp; BENDING STRESS

$$\begin{aligned} M_{x2} &= 43480 \text{ lb-ft} = wdi * (Lsp)^2 / 8 * Km \\ f_{bx2} &= 2195 \text{ psi} = M_{x2} * 12 / S \end{aligned}$$

$$\begin{aligned} M_{y2} &= 0 \\ f_{by2} &= 0 \text{ psi} = M_{y2} * 12 / S \end{aligned}$$

## MIDSPAN DEFLECTION

$$Dely2 \text{ (302)} = 0.079 \text{ in} = wdi * Lsp^4 / 185 / Eb / I * 12^3 / 1000 * Kd$$

SUBJECT

LIGO - BEAM TUBE DESIGN  
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Maximum Differential Settlements

OFFICE: NOE-C REVISION:

REFERENCE NO.

MADE BY RJW	CHKD BY WJC	MADE BY	CHKD BY
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SHT - OF 29

6.11

930212



- CASE 3: Snow Load Reaction, Stresses and Deflections (Hanford, WA)  
FIXED SUPPORT

$$\begin{aligned} Rfx3 &= 0 \\ Rfy3 &= 2908 \text{ lbs} = Kf * 2 * L * wsn \\ Rfz3 &= 0 \end{aligned}$$

## GUIDED SUPPORT

$$\begin{aligned} Rgx3 &= 0 \\ Rgy3 &= 1994 \text{ lbs} = Kg * 2 * L * wsn \end{aligned}$$

## MOMENTS &amp; BENDING STRESS

$$\begin{aligned} Mx3 &= 18020 \text{ lb-ft} = wsn * (Lsp)^2 / 8 * Km \\ fbx3 &= 910 \text{ psi} = Mx3 * 12 / S \end{aligned}$$

$$\begin{aligned} My3 &= 0 \\ fby3 &= 0 \text{ psi} = My3 * 12 / S \end{aligned}$$

## MIDSPAN DEFLECTION

$$Dely3 \text{ (amb)} = 0.031 \text{ in} = wsn * Lsp^4 / 185 / Ea / I * 12^3 / 1000 * Kd$$

- CASE 4: Wind Load Reaction, Stresses and Deflections (Livingston, LA)

## FIXED SUPPORT

$$\begin{aligned} Rfx4 &= 1275 \text{ lbs} = ww * 2 * L * Kf \\ Rfy4 &= 0 \\ Rfz4 &= 0 \end{aligned}$$

## GUIDED SUPPORT

$$\begin{aligned} Rgx4 &= 874 \text{ lbs} = ww * 2 * L * Kg \\ Rgy4 &= 0 \end{aligned}$$

## MOMENTS &amp; BENDING STRESS

$$\begin{aligned} Mx4 &= 0 \\ fbx4 &= 0 \text{ psi} = Mx4 * 12 / S \end{aligned}$$

$$\begin{aligned} My4 &= 7896 \text{ lb-ft} = ww * (Lsp)^2 / 8 * Km \\ fby4 &= 399 \text{ psi} = My4 * 12 / S \end{aligned}$$

## MIDSPAN DEFLECTION

$$Dely4 \text{ (amb)} = 0.014 \text{ in} = ww * Lsp^4 / 185 / Ea / I * 12^3 / 1000 * Kd$$

SUBJECT

LIGO - BEAM TUBE DESIGN  
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Maximum Differential Settlements

OFFICE: NOE-C REVISION:

REFERENCE NO.  
930212

MADE BY	CHKD BY	MADE BY	CHKD BY
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DATE	DATE	DATE	DATE
3/11/94	4-4-94		

SHT ? OF 29

6.12

- CASE 5: Seismic Reaction, Stresses and Deflections (Hanford, WA), X Direction (Horizontal)

## FIXED SUPPORT

Rfx5 = 1287 lbs = wseis \* 2 \* L \* Kf  
Rfy5 = 0  
Rfz5 = 0

## GUIDED SUPPORT

Rgx5 = 882 lbs = wseis \* 2 \* L \* Kg  
Rgy5 = 0

## MOMENTS &amp; BENDING STRESS

Mx5 = 0  
fbx5 = 0 psi = Mx5 \* 12 / S

My5 = 7971 lb-ft = wseis \* (Lsp)<sup>2</sup> / 8 \* Km  
fby5 = 402 psi = My5 \* 12 / S

## MIDSPAN DEFLECTION

Delx5 (amb) = 0.014 in = wseis \* Lsp<sup>4</sup> / 185 / Ea / I \* 12<sup>3</sup> / 1000 \* Kd

- CASE 6: Seismic Reaction, Stresses and Deflections (Hanford, WA), Z Direction (Axial)

## FIXED SUPPORT

Rfx6 = 0  
Rfy6 = 0  
Rfz6 = 2169 lbs = wseis \* 2 \* L

## GUIDED SUPPORT

Rgx6 = 0  
Rgy6 = 0

## MOMENTS &amp; BENDING STRESS

Mx6 = 0  
fbx6 = 0 psi = Mx5 \* 12 / S

My6 = 0  
fby6 = 0 psi = My6 \* 12 / S

## MIDSPAN DEFLECTION

Dely6 (amb) = 0.000 in

SUBJECT

LIGO - BEAM TUBE DESIGN  
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Maximum Differential Settlements

OFFICE: NOE-C

REVISION:

REFERENCE NO.

MADE BY	CHKD BY	MADE BY	CHKD BY
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DATE	DATE	DATE	DATE
3/11/94	4-4-94		

SHT	OF
9	19

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## - CASE 7: Vacuum

Bellows effective area =  $2062.9 \text{ sq in} = \text{PI}()^*(\text{De+Dei})/2)^2/4$ Tube pressure area =  $1866.55 \text{ sq in} = \text{PI}()^*(\text{Di})^2/4$ Bellows pressure area =  $196.35 \text{ sq in} = \text{Bellows effective - tube area}$ Axial force, Pzp =  $2886 \text{ lbs} = 14.7 * \text{Bellows pressure area, pos.} = \text{tension}$ Axial Stress due to vacuum, fav7 =  $148.0 \text{ psi} = \text{Pzp} / \text{A, pos.} = \text{tension}$ 

## FIXED SUPPORT

Rfx7 = 0 lbs

Rfy7 = 0

Rfz7 = 0

## GUIDED SUPPORT

Rgx7 = 0 lbs

Rgy7 = 0

fbx7 = fby7 = 0

Dely7 = 0

SUBJECT

LIGO - BEAM TUBE DESIGN  
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OFFICE: NOE-C REVISION:

REFERENCE NO.  
930212

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3/11/94 4-4-94 DATE

SHT 12 OF 12

6.14

- CASE 8: Differential Settlement of Fixed Support

- Reactions Based on RISA2D with 1" differential settlement

Fixed support = -2.37 kips, per RISA2D

1/2 of guided support = 2.47 kips, per RISA2D

Based on, 1 lbs/ft, DL, used in RISA2D

FIXED SUPPORTS

Rfx8 = -2447 lbs = (RISA2D fixed) - DL \* 2 \* L \* Kf

Rfy8 = -2447 lbs = (RISA2D fixed) - DL \* 2 \* L \* Kf

GUIDED SUPPORTS

Rgx8 = 4887 lbs = 2 \* (RISA2D guided) - DL \* 2 \* L \* Kg

Rgy8 = 4887 lbs = 2 \* (RISA2D guided) - DL \* 2 \* L \* Kg

- Moments based on RISA2D with 1" differential settlement

Maximum Moment = 71360 lb-ft, at fixed support

Based on, 1 lbs/ft, DL + Insulation, used in RISA2D

Mx8 = My8 = 70882 lb-ft = (RISA2D Moment) - DL \* Lsp<sup>2</sup> / 8 \* Km

fbx8 = fby8 = 3579 psi = Mx8 \* 12 / S

SUBJECT

LIGO - BEAM TUBE DESIGN  
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Maximum Differential Settlements

OFFICE: NOE-C REVISION:

REFERENCE NO.  
930212

MADE BY	CHKD BY	MADE BY	CHKD BY
RJW	WJC		
DATE	DATE	DATE	DATE
3/11/94	4-4-94		

SHT 11 OF 29

6.15

## - CASE 9: Expansion Joint Forces at 302 Degrees F.

Maximum Bellows spring rate,  $K = 10062 \text{ lbs/in} = \text{Kej} * (1 + \text{Eej})$ Est. temp. change above 70,  $\text{Tch} = 232 \text{ degrees Fahrenheit}$ Thermal growth hot,  $x = \frac{3.257 \text{ in} = e * 2 * L * 12 * \text{Tch}}$ Longitudinal Comp. force,  $\text{Pbc} = 32776 \text{ lbs} = K * x$  $\text{fa9c} = -1681 \text{ psi} = \text{Pbc} / A$ , stress on tube, neg is comp

## FIXED SUPPORT

 $\text{Rfx9} = 0 \text{ lbs}$  $\text{Rfy9} = 0$  $\text{Rfz9} = 5959 \text{ lbs} = 2 * \text{Eej} * \text{Kej} * x$ 

## GUIDED SUPPORT

 $\text{Rgx9} = 0$  $\text{Rgy9} = 167.8 \text{ lbs} = \text{Rfz9} * H / \text{Lsp} / 12 / 2$  $\text{fbx9} = \text{fby9} = 0$  $\text{Dely9} = \text{Dely9} = 0$ 

## - CASE 10: Expansion Joint Forces at 100 Degrees F.

Maximum Bellows spring rate,  $K = 10062 \text{ lbs/in} = \text{Kej} * (1 + \text{Eej})$ Est. temp. change above 70,  $\text{Tch} = 30 \text{ degrees Fahrenheit}$ Thermal growth hot,  $\text{xw} = \frac{0.421 \text{ in} = e * 2 * L * 12 * \text{Tch}}$ Longitudinal Comp. force,  $\text{Pbc} = 4238 \text{ lbs} = K * \text{xw}$  $\text{fa10c} = -217 \text{ psi} = \text{Pbc} / A$ , stress on tube, neg is comp

## FIXED SUPPORT

 $\text{Rfx10} = 0 \text{ lbs}$  $\text{Rfy10} = 0$  $\text{Rfz10} = 771 \text{ lbs} = 2 * \text{Eej} * \text{Kej} * \text{xw}$ 

## GUIDED SUPPORT

 $\text{Rgx10} = 0$  $\text{Rgy10} = 21.7 \text{ lbs} = \text{Rfz10} * H / \text{Lsp} / 12 / 2$  $\text{fbx10} = \text{fby10} = 0$  $\text{Dely10} = \text{Dely10} = 0$ 

## SUBJECT

LIGO - BEAM TUBE DESIGN  
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REFERENCE NO.  
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MADE BY	CHKD BY	MADE BY	CHKD BY
RJW	WJC		
DATE	DATE	DATE	DATE
3/11/94	3-1-94		

SHT 12 OF 29

6.1.0

## - CASE 11: Expansion Joint Forces at -16 Degrees F.

Maximum Bellows spring rate, K = 10062 lbs/in =  $K_{ej} * (1 + E_{ej})$   
 Est. temp. change below 70, Tchc = 86 degrees Fahrenheit  
 Thermal shrinkage, xs =  $-1.207 \text{ in} = e * 2 * L * 12 * Tchc$   
 Longitudinal Tension force, Pbt = 12150 lbs =  $K * xs$   
 fa11t = 623 psi =  $-Pbt / A$  Positive is tension

## FIXED SUPPORT

Rfx11 = 0 lbs  
 Rfy11 = 0  
 Rfz11 = 2209 lbs =  $-2 * E_{ej} * K_{ej} * xs$

## GUIDED SUPPORT

Rgx11 = 0  
 Rgy11 = 62.2 lbs =  $Rfz11 * H / Lsp / 12 / 2$

fbx11 = fby11 = 0  
 Delx11 = Dely11 = 0

SUBJECT	OFFICE: NOE-C		REVISION:		REFERENCE NO.
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LIGO - BEAM TUBE DESIGN Configuration 3, K = 8316 + 20% Maximum Differential Settlements	DATE 3/11/94	DATE 4.4.94	DATE	DATE	SHT 13 OF 33
					6.17

- LOAD COMBINATIONS FOR MAXIMUM REACTIONS, STRESS AND DEFLECTIONS  
COMBINATION 1 - DL + Insulation + Vacuum + 302 F (Case 2 + 7 + 9)

## FIXED SUPPORTS

$$\begin{aligned} R_x &= 0 \text{ lbs} = R_{fx2} + R_{fx7} + R_{fx9} \text{ (lateral)} \\ R_y &= 7018 \text{ lbs} = R_{fy2} + R_{fy7} + R_{fy9} \text{ (Vertical)} \\ R_z &= 5959 \text{ lbs} = R_{fz2} + R_{fz7} + R_{fz9} \text{ (axial)} \\ R_{\text{max per bolt}} &= 3509 \text{ lbs} = R_y/2 + R_x * H / \text{Abs} \\ R_{\text{min per bolt}} &= 3509 \text{ lbs} = R_y/2 - R_x * H / \text{Abs} \end{aligned}$$

## GUIDED SUPPORTS

$$\begin{aligned} R_x &= 0 \text{ lbs} = R_{gx2} + R_{gx7} + R_{gx9} \\ R_y &= 4980 \text{ lbs} = R_{gy2} + R_{gy7} + R_{gy9} \text{ (Vertical)} \\ R_{\text{max per bolt}} &= 2490 \text{ lbs} = R_y/2 + R_x * H / \text{Abs} \\ R_{\text{min per bolt}} &= 2490 \text{ lbs} = R_y/2 - R_x * H / \text{Abs} \end{aligned}$$

## MIDSPAN DEFLECTION

$$\begin{aligned} \text{Delta x (302)} &= 0.000 \text{ in} = \text{Delx2} + \text{Delx7} + \text{Delx9} \\ \text{Delta y (302)} &= 0.079 \text{ in} = \text{Dely2} + \text{Dely7} + \text{Dely9} \\ \text{Max Delta} &= 0.079 \text{ in} = (\text{Delta x}^2 + \text{Delta y}^2)^{0.5} \end{aligned}$$

## TUBE STRESSES

$$\begin{aligned} f_{ac} &= -1533 \text{ psi, } f_{av7} + f_{a9c}, \text{ neg} = \text{compression} \\ f_{bx} &= 2195 \text{ psi} = f_{bx2} + f_{bx7} + f_{bx9} \\ f_{by} &= 0 \text{ psi} = f_{by2} + f_{by7} + f_{by9} \\ M_d &= 7966 \text{ in-lbs} = f_{ac} * A * (\text{Max Delta} + CE) \\ f_d &= 33.5 \text{ psi} = M_d / S \end{aligned}$$

## COMBINED STRESS (compression)

$$f_{at}/F_a + f_{bx}/F_{bx} + f_{by}/F_{by} + f_d/F_{by} < 1.00$$

Where  $B = F_a = F_{bx} = F_{by} = F_{by}$  = 5900 psi for compression from page 3

$$\begin{aligned} f_{ac} / F_a &= 0.260 \\ f_{bx} / F_{bx} &= 0.372 \\ f_{by} / F_{by} &= 0.000 \\ f_d / F_{by} &= 0.006 \\ \text{SUM} &= 0.638 < 1.00 \end{aligned}$$

Tube is adequate

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	DATE 3/11/94	DATE 4-4-94	DATE	DATE
				SHT 14 OF 25
				6.12

**COMBINATION 2** - DL + Insulation + Settlement + Vacuum + 302 F (Case 2+7+8+9)

## SETTLEMENT WITH COMBINATION

$$\begin{aligned}\text{Delta x, x8} &= 0 \text{ in} & \text{Maxset} &= 0.579 = (x8^2 + y8^2)^{0.5} \\ \text{Delta y, y8} &= 0.579 \text{ in}\end{aligned}$$

## FIXED SUPPORTS

$$\begin{aligned}\text{Rx} &= 0 \text{ lbs} = \text{Rfx2} + \text{Rfx7} + x8 * \text{Rfx8} + \text{Rfx9} \\ \text{Ry} &= 5601 \text{ lbs} = \text{Rfy2} + \text{Rfy7} + y8 * \text{Rfy8} + \text{Rfy9} \\ \text{Rz} &= 5959 \text{ lbs} = \text{Rfz2} + \text{Rfz7} + \text{Rfz9} \\ \text{Rmax per bolt} &= 2800 \text{ lbs} = \text{Ry}/2 + \text{Rx} * \text{H} / \text{Abs} \\ \text{Rmin per bolt} &= 2800 \text{ lbs} = \text{Ry}/2 - \text{Rx} * \text{H} / \text{Abs}\end{aligned}$$

## GUIDED SUPPORTS

$$\begin{aligned}\text{Rx} &= 0 \text{ lbs} = \text{Rgx2} + \text{Rgx7} + x8 * \text{Rgx8} + \text{Rgx9} \\ \text{Ry} &= 7810 \text{ lbs} = \text{Rgy2} + \text{Rgy7} + y8 * \text{Rgy8} + \text{Rgy9} \\ \text{Rmax per bolt} &= 3905 \text{ lbs} = \text{Ry}/2 + \text{Rx} * \text{H} / \text{Abs} \\ \text{Rmin per bolt} &= 3905 \text{ lbs} = \text{Ry}/2 - \text{Rx} * \text{H} / \text{Abs}\end{aligned}$$

## MIDSPAN DEFLECTION

$$\begin{aligned}\text{Delta x (302)} &= 0.000 \text{ in} = 0.5 * x8 \\ \text{Delta y (302)} &= 0.369 \text{ in} = \text{Dely2} + 0.5 * y8 \\ \text{Max Delta} &= 0.369 \text{ in} = (\text{Delta x}^2 + \text{Delta y}^2)^{0.5}\end{aligned}$$

## TUBE STRESSES

$$\begin{aligned}\text{fac} &= -1533 \text{ psi, fav7} + \text{fa9c, neg} = \text{compression} \\ \text{fbx} &= 4267 \text{ psi} = \text{fbx2} + \text{fbx7} + y8 * \text{fbx8} + \text{fbx9} \\ \text{fby} &= 0 \text{ psi} = \text{fby2} + \text{fby7} + x8 * \text{fby8} + \text{fby9} \\ \text{Md} &= 22910 \text{ in-lbs} = \text{fac} * \text{A} * (\text{Max}(\text{Max Delta or maxset}) + \text{CE}) \\ \text{fd} &= 96.4 \text{ psi} = \text{Md} / \text{S}\end{aligned}$$

## COMBINED STRESS (compression)

$$\text{fat}/\text{Fa} + \text{fbx}/\text{Fbx} + \text{fby}/\text{Fby} + \text{fd}/\text{Fbxy} <= 1.00$$

Where B=Fa=Fbx=Fby=Fbxy= 5900 psi for compression from page 3

$$\begin{aligned}\text{fac} / \text{Fa} &= 0.260 \\ \text{fbx} / \text{Fbx} &= 0.723 \\ \text{fby} / \text{Fby} &= 0.000 \\ \text{fd} / \text{Fbxy} &= 0.016 \\ \text{SUM} &= 0.999 < 1.00 & \text{Tube is adequate}\end{aligned}$$

SUBJECT

LIGO - BEAM TUBE DESIGN  
Configuration 3, K = 8316 + 20%  
Maximum Differential Settlements

OFFICE: NOE-C

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MADE BY	CHKD BY	MADE BY	CHKD BY
RJW	WJC		

DATE	DATE	DATE	DATE
3/11/94	4.4.94		

SHT	OF	DATE
15	19	6.19



**COMBINATION 3** - DL + Insul + Seis x + Dif settle + Vac + 302 F (Case 2+5+7+8+9)

**SETTLEMENT WITH COMBINATION**

Delta x, x8 = 0 in      Maxset = 0.788 =  $(x8^2 + y8^2)^{0.5}$   
 Delta y, y8 = 0.788 in

**FIXED SUPPORTS**

Rx = 1287 lbs = Rfx2 + Rfx5 + Rfx7 + x8 \* Rfx8 + Rfx9  
 Ry = 5089 lbs = Rfy2 + Rfy5 + Rfy7 + y8 \* Rfy8 + Rfy9  
 Rz = 5959 lbs = Rfz2 + Rfz5 + Rfz7 + Rfz9  
 Rmax per bolt = 4346 lbs = Ry/2 + Rx \* H / Abs  
 Rmin per bolt = 743 lbs = Ry/2 - Rx \* H / Abs

**GUIDED SUPPORTS**

Rx = 882 lbs = Rgx2 + Rgx5 + Rgx7 + x8\*Rgx8 + Rgx9  
 Ry = 8831 lbs = Rgy2 + Rgy5 + Rgy7 + y8\*Rgy8 + Rgy9  
 Rmax per bolt = 5651 lbs = Ry/2 + Rx \* H / Abs  
 Rmin per bolt = 3180 lbs = Ry/2 - Rx \* H / Abs

**MIDSPAN DEFLECTION**

Delta x (302) = 0.014 in = Delx5 + 0.5 \* x8  
 Delta y (302) = 0.473 in = Dely2 + 0.5 \* y8  
 Max Delta = 0.473 in =  $(\text{Delta } x^2 + \text{Delta } y^2)^{0.5}$

**TUBE STRESSES**

fac = -1533 psi, fav7 + fa9c, neg = compression  
 fbx = 5015 psi = fbx2 + fbx5 + fbx7 + y8 \* fbx8 + fbx9  
 fby = 402 psi = fby2 + fby5 + fby7 + x8 \* fby8 + fby9

Md = 29157 in-lbs = fac \* A \* (Max(Max Delta or maxset) + CE)  
 fd = 122.7 psi = Md / S

**COMBINED STRESS (compression)**

fat/Fa + fbx/Fbx + fby/Fby + fd/Fbxy <= 1.00  
 Where B=Fa=Fbx=Fby=Fbxy= 7080 psi = 1.2 \* B from page 3

fac / Fa = 0.216  
 fbx / Fbx = 0.708  
 fby / Fby = 0.057  
 fd / Fbxy = 0.017  
 SUM = 0.999 < 1.00      Tube is adequate

SUBJECT

LIGO - BEAM TUBE DESIGN  
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 Maximum Differential Settlements

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REFERENCE NO.  
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MADE BY	CHKD BY	MADE BY	CHKD BY
RJW			
DATE	DATE	DATE	DATE
3/11/94			

SHT OF 29

S. L. P.

**COMBINATION 4** - DL + Insul + Seis z + Dif settle + Vac + 302 F (Case 2+5+7+8+9)

## SETTLEMENT WITH COMBINATION

$$\text{Delta x, x8} = \frac{0 \text{ in}}{0.756 \text{ in}} \quad \text{Maxset} = 0.756 = (x8^2 + y8^2)^{0.5}$$

$$\text{Delta y, y8} = \frac{0.756 \text{ in}}{0.756 \text{ in}}$$

## FIXED SUPPORTS

$$\begin{aligned} \text{Rx} &= 0 \text{ lbs} = \text{Rfx2} + \text{Rfx6} + \text{Rfx7} + x8 * \text{Rfx8} + \text{Rfx9} \\ \text{Ry} &= 5168 \text{ lbs} = \text{Rfy2} + \text{Rfy6} + \text{Rfy7} + y8 * \text{Rfy8} + \text{Rfy9} \\ \text{Rz} &= 8128 \text{ lbs} = \text{Rfz2} + \text{Rfz6} + \text{Rfz7} + \text{Rfz9} \\ \text{Rmax per bolt} &= 2584 \text{ lbs} = \text{Ry}/2 + \text{Rx} * \text{H} / \text{Abs} \\ \text{Rmin per bolt} &= 2584 \text{ lbs} = \text{Ry}/2 - \text{Rx} * \text{H} / \text{Abs} \end{aligned}$$

## GUIDED SUPPORTS

$$\begin{aligned} \text{Rx} &= 0 \text{ lbs} = \text{Rgx2} + \text{Rgx6} + \text{Rgx7} + x8 * \text{Rgx8} + \text{Rgx9} \\ \text{Ry} &= 8675 \text{ lbs} = \text{Rgy2} + \text{Rgy6} + \text{Rgy7} + y8 * \text{Rgy8} + \text{Rgy9} \\ \text{Rmax per bolt} &= 4337 \text{ lbs} = \text{Ry}/2 + \text{Rx} * \text{H} / \text{Abs} \\ \text{Rmin per bolt} &= 4337 \text{ lbs} = \text{Ry}/2 - \text{Rx} * \text{H} / \text{Abs} \end{aligned}$$

## MIDSPAN DEFLECTION

$$\begin{aligned} \text{Delta x (302)} &= 0.000 \text{ in} = 0.5 * x8 \\ \text{Delta y (302)} &= 0.457 \text{ in} = \text{Dely27} + 0.5 * y8 \\ \text{Max Delta} &= 0.457 \text{ in} = (\text{Delta x}^2 + \text{Delta y}^2)^{0.5} \end{aligned}$$

## TUBE STRESSES

$$\begin{aligned} \text{fac} &= -1644 \text{ psi, fav7} + \text{fa9c} - \text{Rfz6} / \text{A, neg} = \text{compression} \\ \text{fbx} &= 4901 \text{ psi} = \text{fbx2} + \text{fbx5} + \text{fbx7} + y8 * \text{fbx8} + \text{fbx9} \\ \text{fby} &= 402 \text{ psi} = \text{fby2} + \text{fby5} + \text{fby7} + x8 * \text{fby8} + \text{fby9} \end{aligned}$$

$$\begin{aligned} \text{Md} &= 30247 \text{ in-lbs} = \text{fac} * \text{A} * (\text{Max}(\text{Max Delta or maxset}) + \text{CE}) \\ \text{fd} &= 127.3 \text{ psi} = \text{Md} / \text{S} \end{aligned}$$

## COMBINED STRESS (compression)

$$\begin{aligned} \text{fat/Fa} + \text{fbx/Fbx} + \text{fby/Fby} + \text{fd/Fbxy} &<= 1.00 \\ \text{Where B=Fa=Fbx=Fby=Fbxy} &= 7080 \text{ psi} = 1.2 * \text{B from page 3} \end{aligned}$$

$$\begin{aligned} \text{fac / Fa} &= 0.232 \\ \text{fbx / Fbx} &= 0.692 \\ \text{fby / Fby} &= 0.057 \\ \text{fd / Fbxy} &= \frac{0.018}{0.999} < 1.00 \\ \text{SUM} &= 0.999 < 1.00 \end{aligned}$$

Tube is adequate

SUBJECT

LIGO - BEAM TUBE DESIGN  
Configuration 3, K = 8316 + 20%  
Maximum Differential Settlements

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MADE BY	CHKD BY	MADE BY	CHKD BY
RJW	WJC		
DATE	DATE	DATE	DATE
3/11/94	4.4.94		

SHT 11 OF 25

6.21

**COMBINATION 5** - DL + Wind + Dif Set + 100 F (Case 1 + 4 + 8 + 10)**SETTLEMENT WITH COMBINATION**

$$\begin{aligned} \text{Delta x, x8} &= 0 \text{ in} & \text{Maxset} &= 0.965 = (x8^2 + y8^2)^{0.5} \\ \text{Delta y, y8} &= 0.965 \text{ in} \end{aligned}$$

**FIXED SUPPORTS**

$$\begin{aligned} \text{Rx} &= 1275 \text{ lbs} = \text{Rfx1} + \text{Rfx4} + x8 * \text{Rfx8} + \text{Rfx10} \\ \text{Ry} &= 3413 \text{ lbs} = \text{Rfy1} + \text{Rfy4} + y8 * \text{Rfy8} + \text{Rfy10} \\ \text{Rz} &= 771 \text{ lbs} = \text{Rfz1} + \text{Rfz4} + \text{Rfz10} \\ \text{Rmax per bolt} &= 3491 \text{ lbs} = \text{Ry}/2 + \text{Rx} * \text{H} / \text{Abs} \\ \text{Rmin per bolt} &= -78 \text{ lbs} = \text{Ry}/2 - \text{Rx} * \text{H} / \text{Abs} \end{aligned}$$

**GUIDED SUPPORTS**

$$\begin{aligned} \text{Rx} &= 874 \text{ lbs} = \text{Rgx1} + \text{Rgx4} + x8 * \text{Rgx8} + \text{Rgx10} \\ \text{Ry} &= 8697 \text{ lbs} = \text{Rgy1} + \text{Rgy4} + y8 * \text{Rgy8} + \text{Rgy10} \\ \text{Rmax per bolt} &= 5572 \text{ lbs} = \text{Ry}/2 + \text{Rx} * \text{H} / \text{Abs} \\ \text{Rmin per bolt} &= 3125 \text{ lbs} = \text{Ry}/2 - \text{Rx} * \text{H} / \text{Abs} \end{aligned}$$

**MIDSPAN DEFLECTION**

$$\begin{aligned} \text{Delta x (302)} &= 0.014 \text{ in} = \text{Dex4} + 0.5 * x8 \\ \text{Delta y (302)} &= 0.545 \text{ in} = \text{Dey1} + 0.5 * y8 \\ \text{Max Delta} &= 0.545 \text{ in} = (\text{Delta x}^2 + \text{Delta y}^2)^{0.5} \end{aligned}$$

**TUBE STRESSES**

$$\begin{aligned} \text{fac} &= -217 \text{ psi, fav4} + \text{fa10c, neg} = \text{compression} \\ \text{fbx} &= 5260 \text{ psi} = \text{fbx1} + \text{fbx4} + y8 * \text{fbx8} + \text{fbx10} \\ \text{fby} &= 399 \text{ psi} = \text{fby1} + \text{fby4} + x8 * \text{fby8} + \text{fby10} \end{aligned}$$

$$\begin{aligned} \text{Md} &= 4885 \text{ in-lbs} = \text{fac} * \text{A} * (\text{Max}(\text{Max Delta or maxset}) + \text{CE}) \\ \text{fd} &= 20.6 \text{ psi} = \text{Md} / \text{S} \end{aligned}$$

**COMBINED STRESS (compression)**

$$\text{fat}/\text{Fa} + \text{fbx}/\text{Fbx} + \text{fby}/\text{Fby} + \text{fd}/\text{Fbxy} < = 1.00$$

Where B=Fa=Fbx=Fby=Fbxy= 5900 psi for compression from page 3

$$\begin{aligned} \text{fac} / \text{Fa} &= 0.037 \\ \text{fbx} / \text{Fbx} &= 0.891 \\ \text{fby} / \text{Fby} &= 0.068 \\ \text{fd} / \text{Fbxy} &= 0.003 \end{aligned}$$

$$\text{SUM} = 0.999 < 1.00 \quad \text{Tube is adequate}$$

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	RJW	WJC		SHT 19 OF 29
	DATE	DATE	DATE	DATE
	3/11/94	4-4-94		4.22

**COMBINATION 6** - DL + Insulation + Vacuum + 100 F (Case 2 + 7 + 10)**FIXED SUPPORTS**

$$\begin{aligned} R_x &= 0 \text{ lbs} = R_{fx2} + R_{fx7} + R_{fx10} \text{ (lateral)} \\ R_y &= 7018 \text{ lbs} = R_{fy2} + R_{fy7} + R_{fy10} \text{ (Vertical)} \\ R_z &= 771 \text{ lbs} = R_{fz2} + R_{fz7} + R_{fz10} \text{ (axial)} \\ R_{\text{max per bolt}} &= 3509 \text{ lbs} = R_y/2 + R_x * H / \text{Abs} \\ R_{\text{min per bolt}} &= 3509 \text{ lbs} = R_y/2 - R_x * H / \text{Abs} \end{aligned}$$

**GUIDED SUPPORTS**

$$\begin{aligned} R_x &= 0 \text{ lbs} = R_{gx2} + R_{gx7} + R_{gx10} \\ R_y &= 4834 \text{ lbs} = R_{gy2} + R_{gy7} + R_{gy10} \text{ (Vertical)} \\ R_{\text{max per bolt}} &= 2417 \text{ lbs} = R_y/2 + R_x * H / \text{Abs} \\ R_{\text{min per bolt}} &= 2417 \text{ lbs} = R_y/2 - R_x * H / \text{Abs} \end{aligned}$$

**MIDSPAN DEFLECTION**

$$\begin{aligned} \text{Delta x (302)} &= 0.000 \text{ in} = \text{Delx2} + \text{Delx7} + \text{Delx10} \\ \text{Delta y (302)} &= 0.079 \text{ in} = \text{Dely2} + \text{Dely7} + \text{Dely10} \\ \text{Max Delta} &= 0.079 \text{ in} = (\text{Delta x}^2 + \text{Delta y}^2)^{0.5} \end{aligned}$$

**TUBE STRESSES**

$$\begin{aligned} f_{ac} &= -69 \text{ psi, } f_{av7} + f_{a10c}, \text{ neg} = \text{compression} \\ f_{bx} &= 2195 \text{ psi} = f_{bx2} + f_{bx7} + f_{bx10} \\ f_{by} &= 0 \text{ psi} = f_{by2} + f_{by7} + f_{by10} \end{aligned}$$

$$\begin{aligned} M_d &= 360 \text{ in-lbs} = f_{ac} * A * (\text{Max Delta} + CE) \\ f_d &= 1.5 \text{ psi} = M_d / S \end{aligned}$$

**COMBINED STRESS (compression)**

$$f_{at}/F_a + f_{bx}/F_{bx} + f_{by}/F_{by} + f_d/F_{bxy} <= 1.00$$

Where  $B = F_a = F_{bx} = F_{by} = F_{bxy} = 5900$  psi for compression from page 3

$$\begin{aligned} f_{ac} / F_a &= 0.012 \\ f_{bx} / F_{bx} &= 0.372 \\ f_{by} / F_{by} &= 0.000 \\ f_d / F_{bxy} &= 0.000 \\ \text{SUM} &= 0.384 < 1.00 \end{aligned}$$

Tube is adequate

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MADE BY R JW	CHKD BY WJC	MADE BY	CHKD BY
DATE 3/11/94	DATE 4-4-94	DATE	DATE

SHT 1 <sup>A</sup> OF 2 <sup>A</sup>	v. 23
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