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function[pend] = quadopt
% Design of quad pendulum with blade spring optimisation v1.01
% Written by KAS 7/99
% edited by CIT 11/99
% changed again by KAS 12/99
% changes to use structure 12/99 KAS
% Calum's/Mike's/Matts blade equations added 12/99 KAS
% routine to optimise blades added 12/99 KAS
% pendulum updated 27/1/00 CIT & KAS
% QUAD PENDULUM MODEL WITH CANTILEVERS
% see pendulum diagram for definitions of parameters

% the parameters are now fixed to represent the REFERENCE DESIGN
% for the LIGO II main optics suspension
%
% coordinates x = longitudinal = u_LIGO roll about this axis
% y = transverse = v_LIGO pitch about this axis
% z = vertical = w_LIGO yaw about this axis

%*****%
% MASS (N) is REPRESENTED by a rectangular BLOCK
% in reality it will be larger and less dense.
global pend
pend.g      = 9.81;
pend.nx = 0.125;        %dimensions of MASS (N) (square)
pend.ny = 0.40;
pend.nz = 0.13;
pend.denn = 5600;        %density (steel)
pend.mn = pend.denn* pend.ny* pend.nz* pend.nx;      %MASS (N)
pend.Inx = pend.mn*( pend.ny^2+ pend.nz^2)/12; %moment of inertia (transverse roll)
pend.Iny = pend.mn*( pend.nz^2+ pend.nx^2)/12; %moment of inertia (longitudinal pitch)
pend.Inz = pend.mn*( pend.ny^2+ pend.nx^2)/12; %moment of inertia (yaw)

%*****%
% MASS (1) is REPRESENTED by a rectangular BLOCK
% in reality it will be larger and less dense.

pend.ux = 0.125;        %dimensions MASS (1) (square)
pend.uy = 0.40;
pend.uz = 0.13;
pend.den1 = 5600;        %density (steel)

pend.m1 = pend.den1* pend.uy* pend.uz* pend.ux ;  %MASS (1)

pend.I1x = pend.m1*( pend.uy^2+ pend.uz^2)/12; %moment of inertia (transverse roll)
pend.I1y = pend.m1*( pend.uz^2+ pend.ux^2)/12; %moment of inertia (longitudinal pitch)
pend.I1z = pend.m1*( pend.uy^2+ pend.ux^2)/12; %moment of inertia (yaw)

%*****%
pend.ix = 0.125;        %dimension of MASS (2) (cylinder)
pend.ir = 0.16;
pend.den2 = 5600;
pend.m2 = pend.den2*pi* pend.ir^2* pend.ix;      %MASS (2)

pend.I2x = pend.m2*(pend.ir^2/2);                  %moment of inertia (transverse roll)
pend.I2y = pend.m2*(pend.ir^2/4+pend.ix^2/12);    %moment of inertia (longitudinal pitch)

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pend.I2z = pend.m2*(pend.ir^2/4+pend.ix^2/12); %moment of inertia (yaw)

%*****
pend.tx = 0.125; %dimensions of MASS (3) (cylinder)
pend.tr = 0.16;
pend.den3 = 4000; %density (sapphire)

pend.m3 = pend.den3*pi*pend.tr^2*pend.tx; %MASS (3)

pend.I3x = pend.m3*(pend.tr^2/2); %moment of inertia (transverse roll)
pend.I3y = pend.m3*(pend.tr^2/4+pend.tx^2/12); %moment of inertia (longitudinal pitch)
pend.I3z = pend.m3*(pend.tr^2/4+pend.tx^2/12); %moment of inertia (yaw)

%*****
pend.ln = 0.54; %wire length 1: short version
pend.l1 = 0.304; %wire length 2: short version
pend.l2 = 0.302; %wire length 3: short version
pend.l3 = 0.6; %wire length 4

%*****
pend.nwn = 2; %number of wires (= number of cantilevers if fitted) per stage (2 or 4)
pend.nw1 = 4;
pend.nw2 = 4;
pend.nw3 = 4;

%*****
pend.rn = 800e-6; %radius of wire (N)
pend.r1 = 805e-6; %radius of wire (1)
pend.r2 = 602e-6; %radius of wire (2)
pend.r3 = 200e-6; %radius of wire (3)

%*****
pend.Yn = 1.65e11; %Youngs Modulus of wire (N) (s/steel 302)
pend.Y1 = 1.65e11; %Youngs Modulus of wire (1) (s/steel 302)
pend.Y2 = 1.65e11; %Youngs Modulus of wire (2) (s/steel 302)
pend.Y3 = 7e10; %Youngs Modulus of wire (3) (fused silica)

%blade design
mntb = (pend.mn+pend.m1+pend.m2+pend.m3)/2;%total per blade
mnb = pend.mn/2;%uncoupled mass
[uf,lnb,anb,hnb,stn] = opt(mnb,mntb,9e8,0.55,0.12);
pend.lnb = lnb;
pend.anb = anb;
pend.hnb = hnb;
pend.ufcn = uf;
pend.stn = stn;
pend.intmode_n = 55*hnb*0.37^2/(0.002*lnb^2); %scaled from GEO blade

%blade design
mltb = (pend.m1+pend.m2+pend.m3)/2;%total per blade
mlb = pend.m1/2;%uncoupled mass
[uf,l1b,a1b,h1b,st1] = opt(mlb,mltb,8.5e8,0.48,0.07);
pend.l1b = l1b;
pend.a1b = a1b;
pend.h1b = h1b;

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pend.ufc1 = uf;
pend.st1 = st1;
pend.intmode_1 = 55*h1b*0.37^2/(0.002*l1b^2); %scaled from GEO blade
%blade design
m2tb = (pend.m2+pend.m3)/2;%total per blade
m2b = pend.m2/2;%uncoupled mass
[uf,l2b,a2b,h2b,st2] = opt(m2b,m2tb,8e8,0.4,0.07);
pend.l2b = l2b;
pend.a2b = a2b;
pend.h2b = h2b;
pend.ufc2 = uf;
pend.st2 = st2;
pend.intmode_2 = 55*h2b*0.37^2/(0.002*l2b^2); %scaled from GEO blade

*****%
dees = 0.001;
pend.dm = dees; %height of wire break-off above c.of m. mass (N)
pend.dn = dees; %height of wire break-off below c.of m. mass (N)
pend.d0 = dees; %height of wire break-off above c.of m. mass (1)
pend.d1 = dees; %height of wire break-off below c.of m. mass (1)
pend.d2 = dees; %height of wire break-off above c.of m. mass (2)
pend.d3 = dees; %height of wire break-off below c.of m. mass (2)
pend.d4 = dees; %height of wire break-off above c.of m. mass (3)
%additional information for ribbon breakoffs
%these are needed in translation/roll mode calculations
pend.twistlength = 0.006; %length of twist section in ribbon
pend.d3tr = dees - pend.twistlength;
pend.d4tr = dees - pend.twistlength;
*****%
%X direction separation
pend.sn = 0.00; %1/2 separation of wires (N)
pend.su = 0.002; %1/2 separation of wires (1)
pend.si = 0.002; %1/2 separation of wires (2)
pend.sl = 0.007; %1/2 separation of wires (3)

*****%
%Y direction separation
pend.nn0 = 0.25; %1/2 separation of wires (N) at suspension point
pend.nn1 = 0.05; %1/2 separation of wires (N) at mass (N)
pend.n0 = 0.2; %1/2 separation of wires (1) at mass (N)
pend.n1 = 0.07;
pend.n2 = 0.12;
pend.n3 = pend.ir+0.0065; %1/2 separation of wires (2) at mass (2)
pend.n4 = pend.tr+0.0015; %1/2 separation of wires (3) at mass (2)
pend.n5 = pend.tr+0.0015; %1/2 separation of wires (3) at mass (3)

%now we can work out the true lengths com to com etc.
pend.tln = sqrt(pend.ln^2 - (pend.nn0-pend.nn1)^2);
pend.tl1 = sqrt(pend.l1^2 - (pend.n0-pend.n1)^2);
pend.tl2 = sqrt(pend.l2^2 - (pend.n2-pend.n3)^2);
pend.tl3 = sqrt(pend.l3^2 - (pend.n4-pend.n5)^2);
pend.l_total = pend.tln +pend.tl1+pend.tl2+pend.tl3;

*****%
% represents small loss
pend.bd = 0.01; % makes phases of open loop plots look nicer

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pend.lever_roll = 0.15;  
pend.lever_pitch = 0.05;  
pend.lever_yaw = 0.15;  
  
%end
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